

# **RANS Modeling of Stably Stratified Turbulent Boundary Layer Flows in OpenFOAM**

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**Colorado  
State  
University**



# Talk Outline

- Motivation
- Theoretical Background
- Numerical Setup
- Simulation Results
- Conclusions
- Future Research

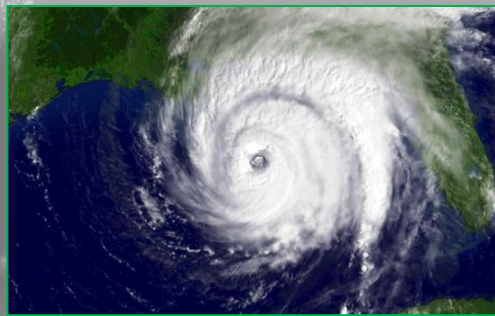


# Motivation

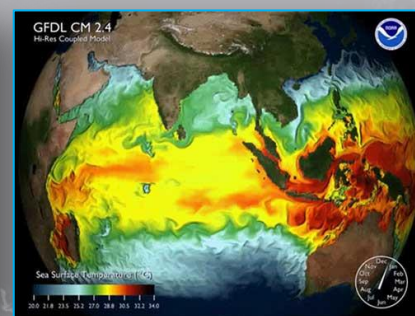
- The atmospheric boundary layer (ABL) is a very turbulent flow  $Re \sim O(10^7)$
- Turbulent processes influence the transport of momentum, heat, humidity, and scalars.



Wind Energy



Weather



Climate Change

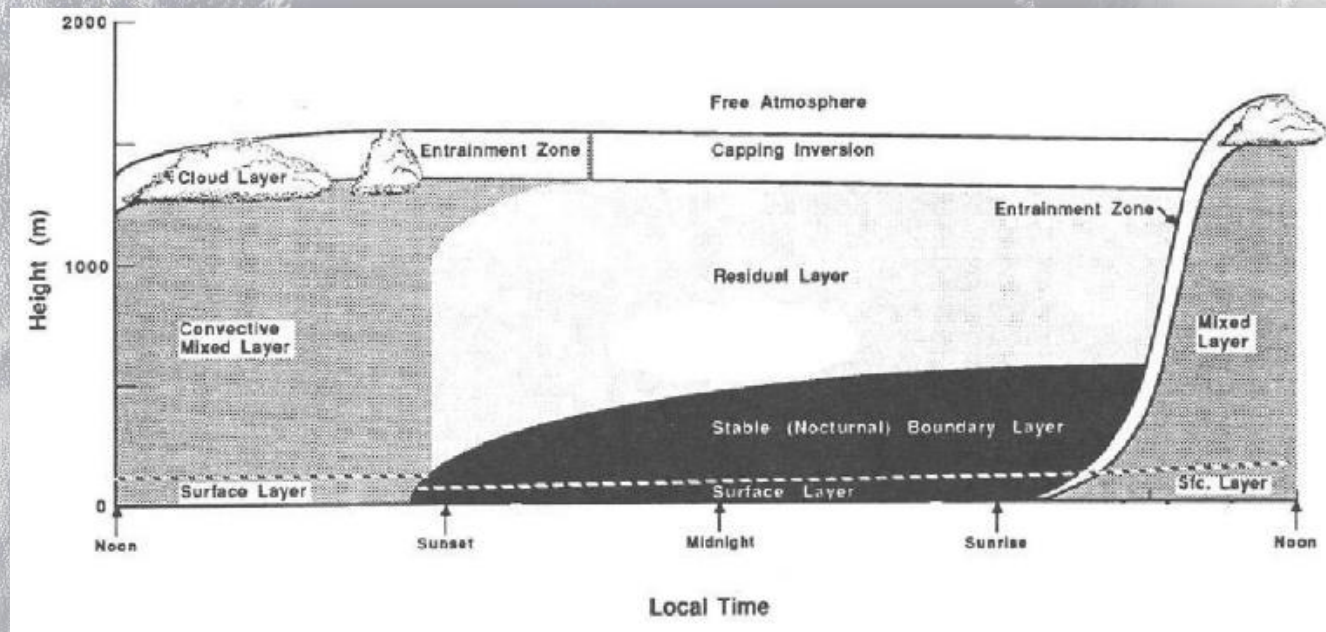


Air Quality



# Motivation

- The ABL experiences a diurnal cycle ( $\sim 24$  hr) due to the radiative heating and cooling of the earth's surface

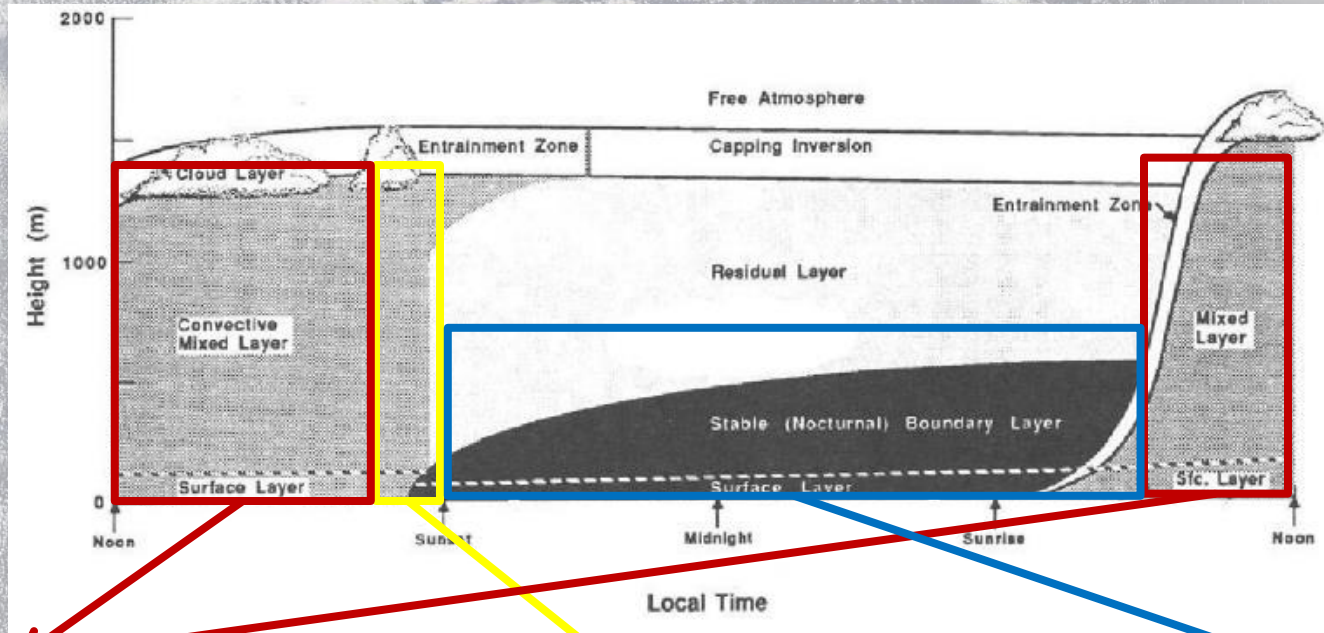


Stull  
(1988)



# Motivation

Stull  
(1988)



## Unstable ABL

- Positive surface heat flux
- Mixing is enhanced by convection
- Surface, free convection, and mixed layers

## Neutral ABL

- Uniform temperature distribution
- Logarithmic wind profile
- Relatively well understood

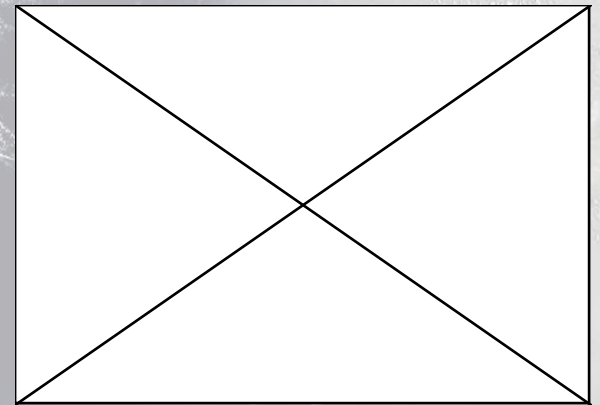
## Stable ABL

- Negative surface heat flux
- Mixing is suppressed resulting in a shallow layer
- Exhibit inertial phenomena (e.g. jets and waves)



# Motivation

- The Stable Atmospheric Boundary Layer (SABL)
  - Buoyancy forces lead to:
    - Wave formation and breaking (intermittent turbulence)
    - Low-level jets
    - Suppressed turbulent scales
  - These effects present significant challenges for wind energy
    - Increased shear forces and fatigue loading on wind turbines





# Theoretical Background

- **Reynolds-Averaged Navier-Stokes (RANS) Framework**

- **Continuity**

- $\frac{\partial \bar{U}_j}{\partial x_i} = 0$

- **Momentum**

- $\frac{D\bar{U}_j}{Dt} = -\frac{1}{\rho_0} \frac{\partial \bar{p}}{\partial x_i} + \frac{\partial}{\partial x_i} \left[ \nu_{\text{eff}} \left( \frac{\partial \bar{U}_i}{\partial x_j} + \frac{\partial \bar{U}_j}{\partial x_i} \right) \right] + g\delta_{ij3} + 2\Omega_{ij} \times \bar{U}_{ij} + F_j$

- $\nu_{\text{eff}} = \nu + \nu_t$

- **Scalar (Density) Transport**

- $\frac{D\bar{\rho}}{Dt} = \frac{\partial}{\partial x_i} \left( \kappa_{\text{eff}} \frac{\partial \bar{\rho}}{\partial x_i} \right)$

- $\kappa_{\text{eff}} = \kappa + \kappa_t \Rightarrow \kappa_t = \frac{\nu_t}{Pr_t}$



# Theoretical Background

- **Quantifying Stability**

- **Gradient Richardson Number ( $Ri$ )**

- $Ri = \frac{N^2}{S^2}$

- Where  $N = \sqrt{-\left(\frac{g}{\rho_0}\right)\frac{d\bar{\rho}}{dz}}$  is the buoyancy frequency and  $S = \frac{d\bar{u}}{dz}$  is the mean shear

- **Flux Richardson Number ( $Ri_f$ )**

- $Ri_f = \frac{-B}{P}$

- Where  $B = -\left(\frac{g}{\rho_0}\right)\overline{w'\rho'} = \kappa_t\left(\frac{g}{\rho_0}\right)\frac{d\bar{\rho}}{dz}$  is the buoyancy production and  $P = -\overline{u'w'}S = \nu_t S^2$  is the shear production

- **Monin-Obukhov length**

- $L = -\frac{u_*^3}{\left(\frac{g}{\rho_0}\right)\kappa(\overline{w'\rho'})_s}$



# Theoretical Background

- **k – ε Turbulence Model**

- Shown to perform well for stably stratified geophysical flows (e.g. Rodi 1987; Baumert & Peters 2000) and SABL (Detering & Etling 1985; Apsley & Castro 1997)

- **Turbulent kinetic energy (k)**

- $$\frac{\overline{D}k}{\overline{D}t} = \frac{\partial}{\partial x_i} \left( \frac{v_t}{\sigma_k} \frac{\partial k}{\partial x_i} \right) + P + B - \varepsilon$$

- **Dissipation rate of turbulent kinetic energy (ε)**

- $$\frac{\overline{D}\varepsilon}{\overline{D}t} = \frac{\partial}{\partial x_i} \left( \frac{v_t}{\sigma_\varepsilon} \frac{\partial \varepsilon}{\partial x_i} \right) + C_{\varepsilon 1} (P + C_{\varepsilon 3} B) \frac{\varepsilon}{k} - C_{\varepsilon 2} \frac{\varepsilon^2}{k}$$

- **Turbulent viscosity (v<sub>t</sub>)**

- $$v_t = (1 - Ri_f) C_\mu \frac{k^2}{\varepsilon}$$

$\sigma_k$	$\sigma_\varepsilon$	$C_\mu$	$C_{\varepsilon 1}$	$C_{\varepsilon 2}$	$C_{\varepsilon 3}$
1.0	1.3	0.09	1.44	1.92	-1.44



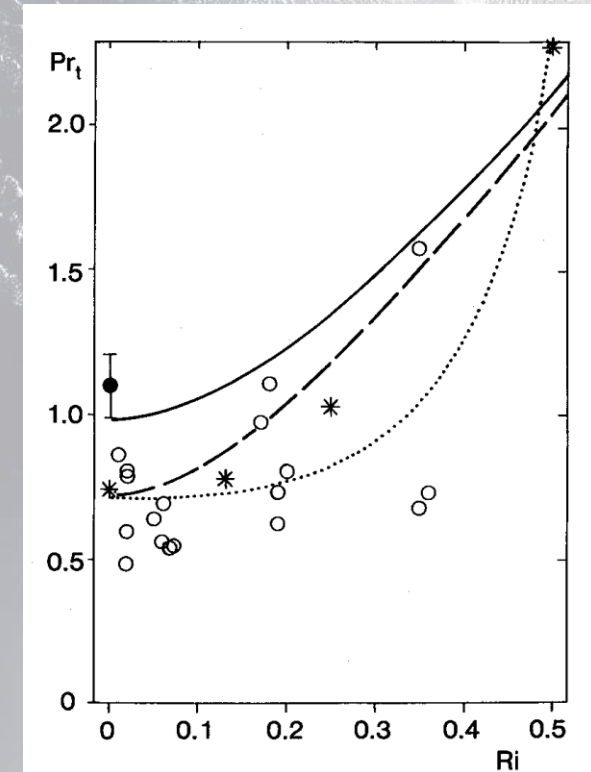
# Theoretical Background

- **The Turbulent Prandtl Number ( $Pr_t$ )**
    - Links eddy viscosity and diffusivity in a RANS framework
    - In modeling, generally assumed to be a constant ( $Pr_t = 0.85$ ; e.g. Wilcox 1994)
    - However, research has shown that  $Pr_t$  is strongly linked to stability for stable stratification
    - Firstly, we will consider 3  $Pr_t$  formulations
      - A constant value:  $Pr_t = 0.85$
      - Kim & Mahrt (1992) develop a  $Pr_t$  from the Louis (1981) model given by:
$$Pr_t = \frac{l_{0,m}^2 \phi_m^2(Ri)}{l_{0,h}^2 \phi_h^2(Ri)}$$
- $\Rightarrow$   $Pr_t = Pr_{t0} \frac{1+15Ri(1+5Ri)^{1/2}}{1+10Ri(1+5Ri)^{-1/2}} \quad \text{(KM92)}$



# Theoretical Background

- **The Turbulent Prandtl Number ( $Pr_t$ )**
  - Venayagamoorthy & Stretch (2010) developed a  $Pr_t$  formulation for stably stratified homogeneous shear flows from the empirical model of Schumann & Gerz (1995)
  - Sought to generalize a Prandtl number formulation to include irreversible contributions and the neutral value of  $Pr_t$
  - For the weakly stratified regime ( $Ri \lesssim 0.25$ )
    - $Pr_t = Pr_{t0} + Ri$
  - For the strongly stratified regime (large  $Ri$ )
    - $Pr_t = \frac{1}{Ri_{f\infty}} Ri$
  - Fitting a blending function between these two regimes results in:
    - $Pr_t = Pr_{t0} \exp\left(-\frac{Ri}{Pr_{t0}\Gamma_\infty}\right) + \frac{Ri}{Ri_{f\infty}}$  (VS10)
      - $Pr_{t0} \simeq 0.7$
      - $\Gamma = \frac{Ri_f}{(1-Ri_f)} \Rightarrow Ri_{f\infty} \simeq 0.25 \Rightarrow \Gamma_\infty = 1/3$



Schumann & Gerz (1995)



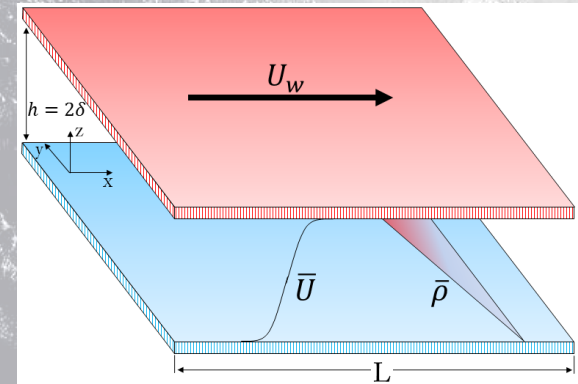
# Numerical Setup

- Starting with 2 simplified flow cases
  - Stably stratified turbulent Couette flow**

- Based on the DNS work of Garcia-Villalba et al. (2011a)

- $Re_\tau = 540, Ri_\tau = 83.5$
  - $Re_\tau = 540, Ri_\tau = 167$

$$Re_\tau = \frac{u_\tau \delta}{\nu}$$

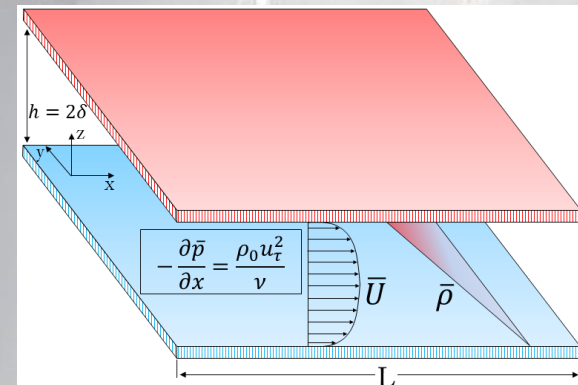


- Stably stratified turbulent channel flow**

- Based on the DNS work of Garcia-Villalba & del Alamo (2011b)

- $Re_\tau = 550, Ri_\tau = 60$
  - $Re_\tau = 550, Ri_\tau = 120$

$$Ri_\tau = \frac{\Delta \rho g h}{\rho_0 u_\tau^2}$$





# Numerical Setup

- OpenFOAM
  - Couette flow

```
tmp<fvVectorMatrix> UEqn
(
    fvm::div(phi, U)
    + turbulence->divDevReff(U)
    ==
    sources(U)
);

UEqn.relax();

sources.constrain(UEqn());

UEqn().solve();
```



# Numerical Setup

- OpenFOAM
  - Channel flow

```
tmp<fvVectorMatrix> UEqn
(
    fvm::div(phi, U)
    + turbulence->divDevReff(U)
    ==
    sources(U)
);

UEqn().relax();

sources.constrain(UEqn());

solve(UEqn() == -gradP);
```



# Numerical Setup

- OpenFOAM

- Density transport equation

```
kappat = turbulence->nut()/Prt;  
kappat.correctBoundaryConditions();  
volScalarField kappaEff("kappaEff", turbulence->nu()/Pr + kappat);
```

```
fvScalarMatrix rhoEqn  
(  
    fvm::div(phi, rho)  
    -fvm::laplacian(kappaEff, rho)  
);
```

```
rhoEqn.relax();  
rhoEqn.solve();
```



# Numerical Setup

## OpenFOAM

### Modified $k - \varepsilon$ turbulence model

#### Lookup mean density field

```
rho.db().lookupObject<volScalarField>("rho");
```

#### Calculate density gradient

```
gradRho_ = fvc::grad(rho_);  
gradrho_ = gradRho_.component(2);
```

#### Calculate buoyancy frequency (squared)

```
N2_ = (-g_/rho0_)*gradrho_;
```

#### Calculate gradient Richardson number

```
Rig_ = (N2_/(2.0*magSqr(symm(fvc::grad(U_)))));
```

#### Calculate buoyancy production

```
B_ = (nut_/Prt_)*(g_/rho0_)*grahrho_;
```

#### Calculate flux Richardson number

```
Rif_ = min(1.0,-B_/max(Gk_,epsilonMin_));
```

#### Modified $\varepsilon$ equation

```
... C1_*(G + C3_*B))*epsilon_/k_ ...
```

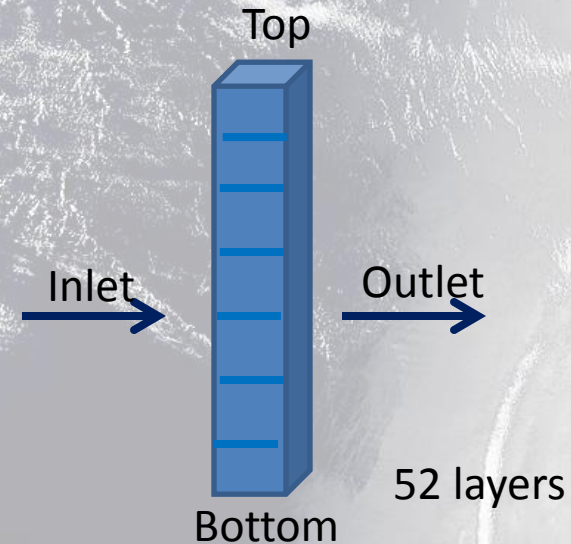
#### Modified $\nu_t$ equation

```
nut_ = (1.0 - Rif_)*Cmu_*sqr(k_)/epsilon_;
```



# Numerical Setup

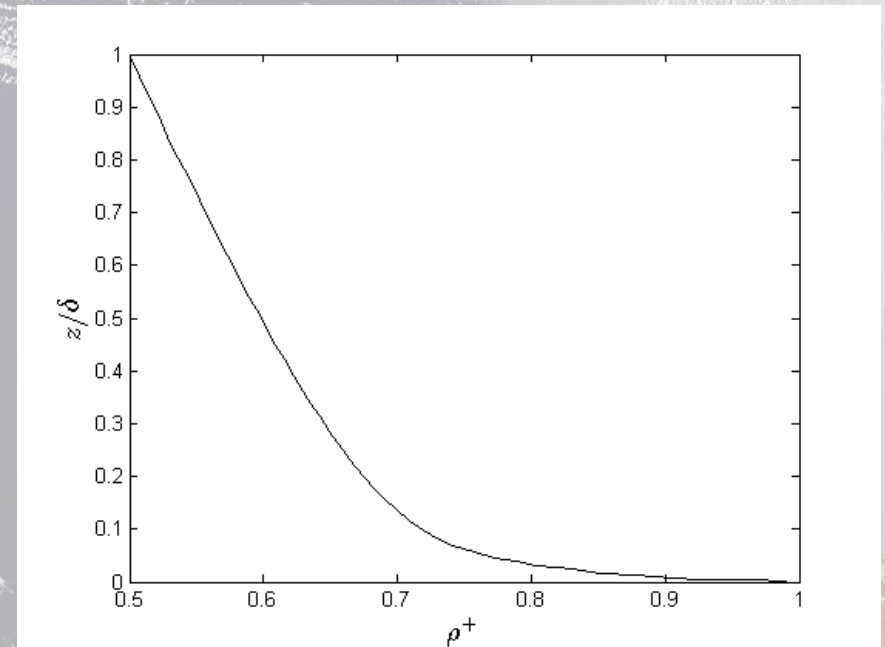
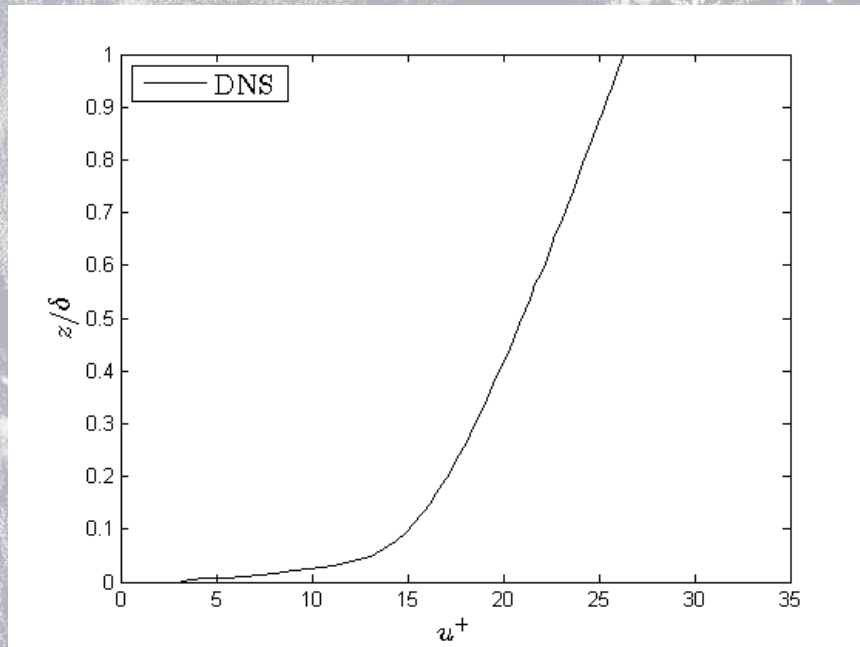
- OpenFOAM
  - Boundary conditions
    - Inlet and outlet – cyclic
    - Front and back – empty
    - Top and bottom – walls
  - Standard wall functions
    - nutkWallFunction
    - kqRWallFunction
    - epsilonWallFunction
  - Second-order accuracy discretization





# Couette Flow Results

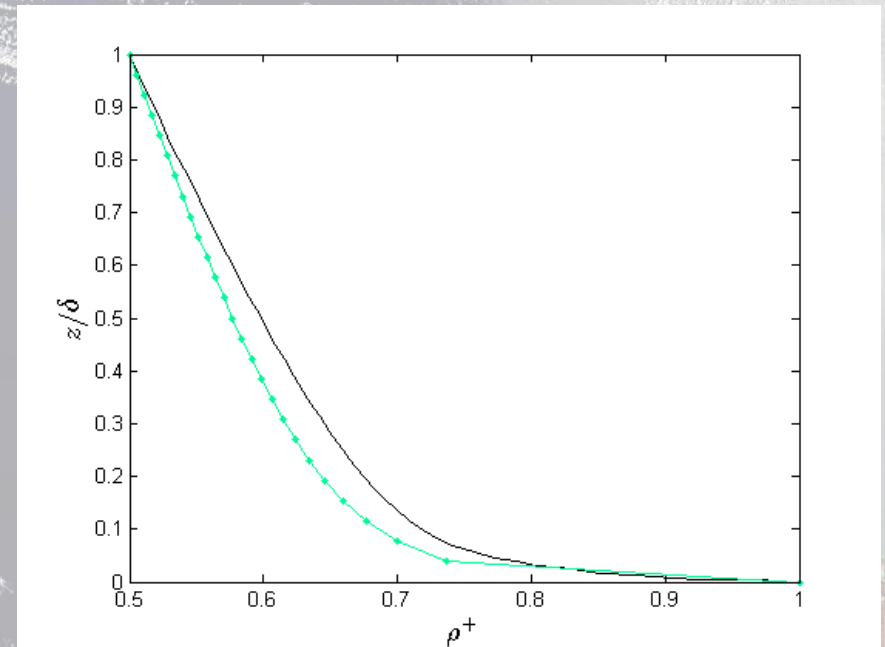
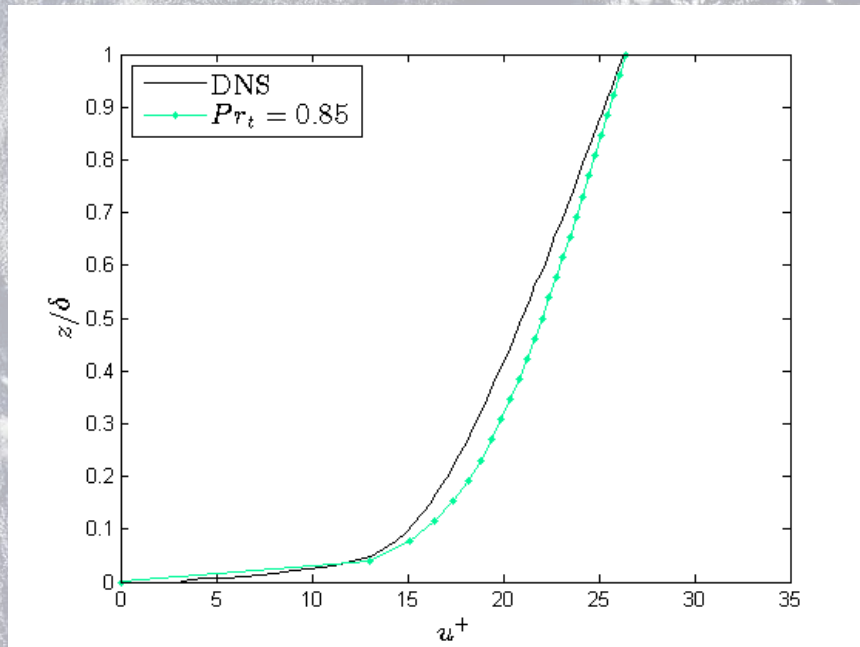
- $Ri_\tau = 83.5$





# Couette Flow Results

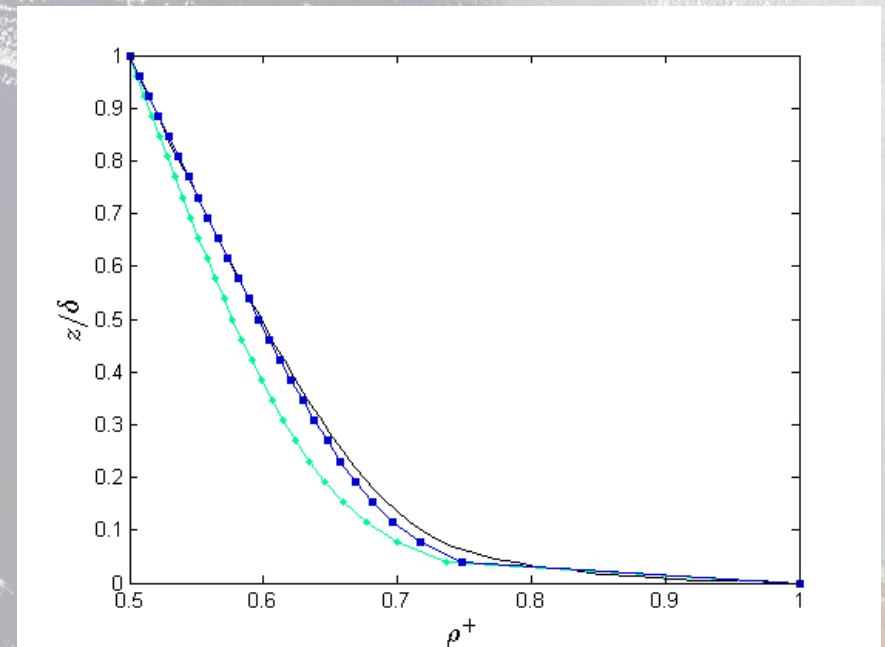
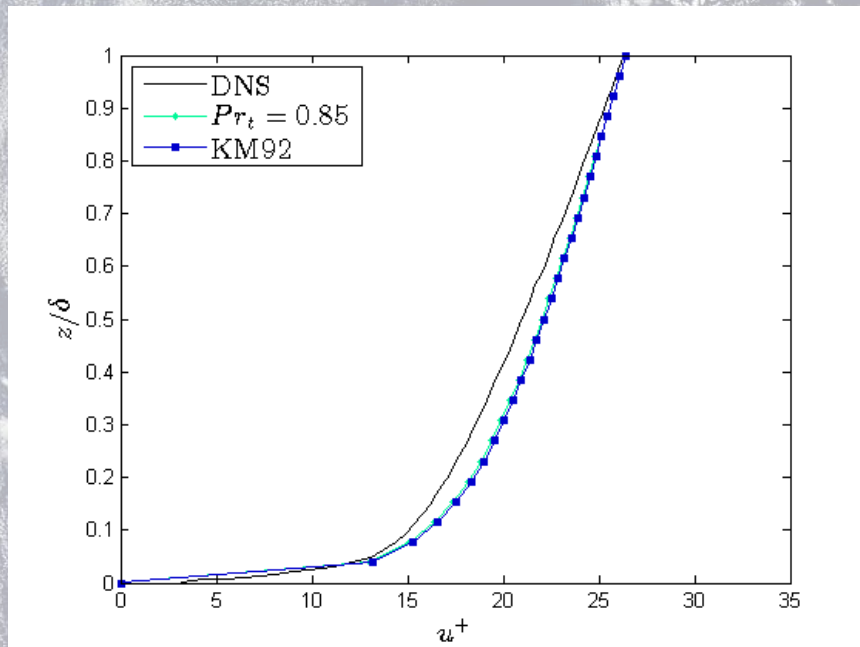
- $Ri_\tau = 83.5$





# Couette Flow Results

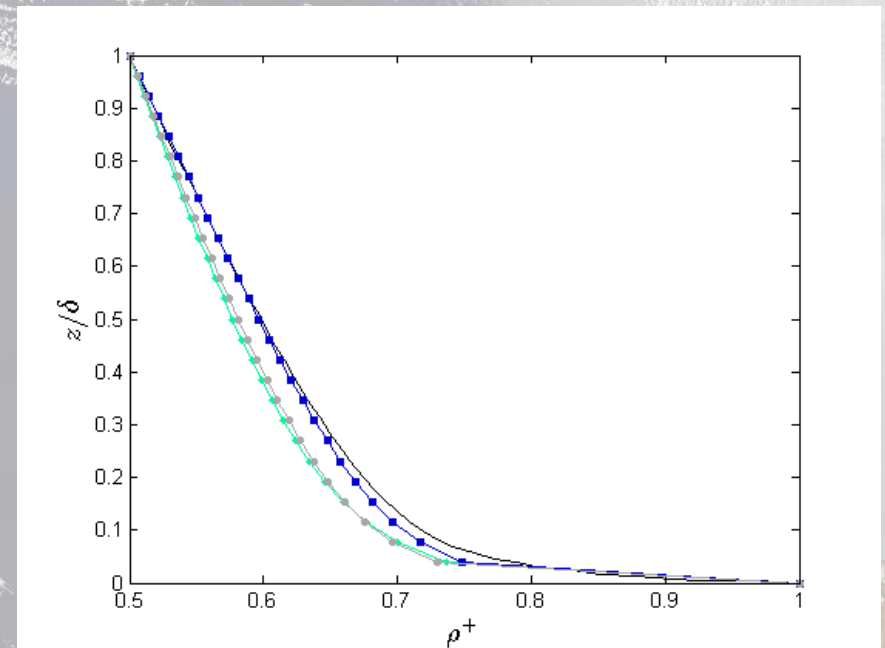
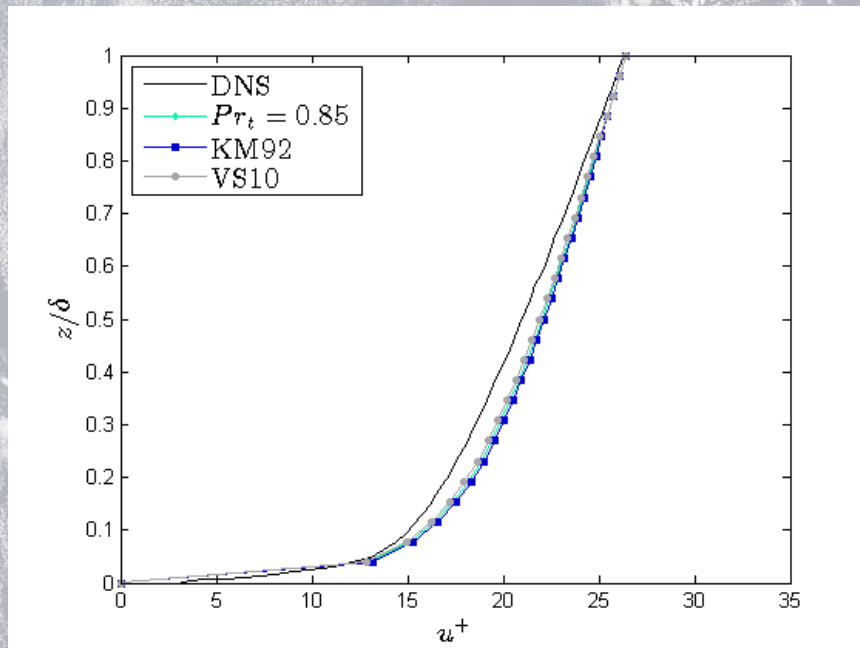
- $Ri_\tau = 83.5$





# Couette Flow Results

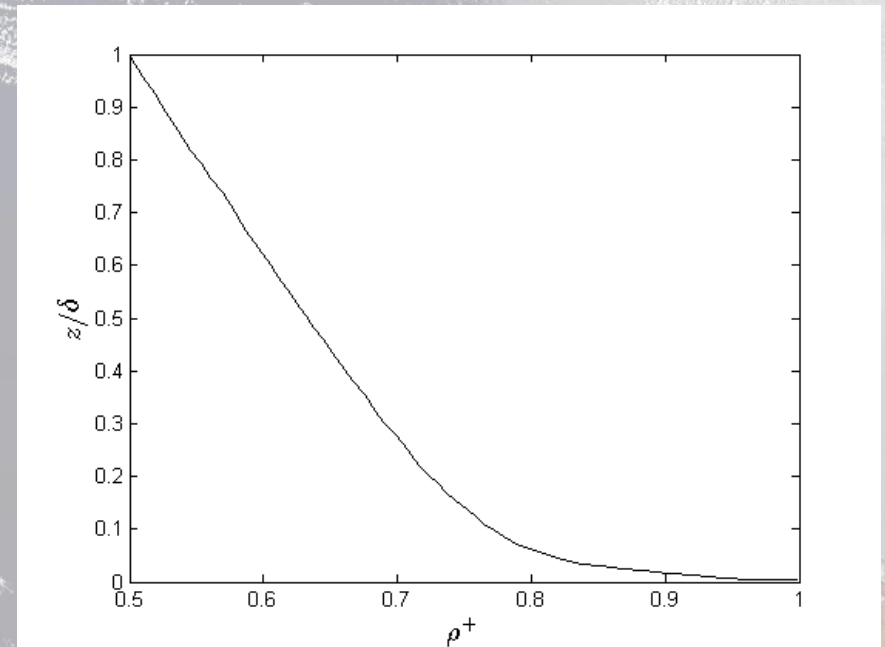
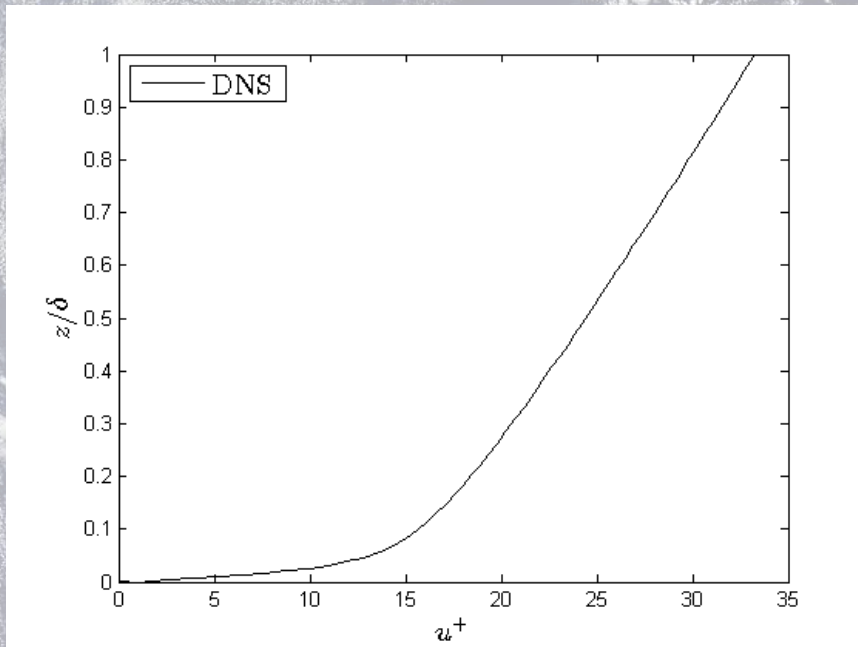
- $Ri_\tau = 83.5$





# Couette Flow Results

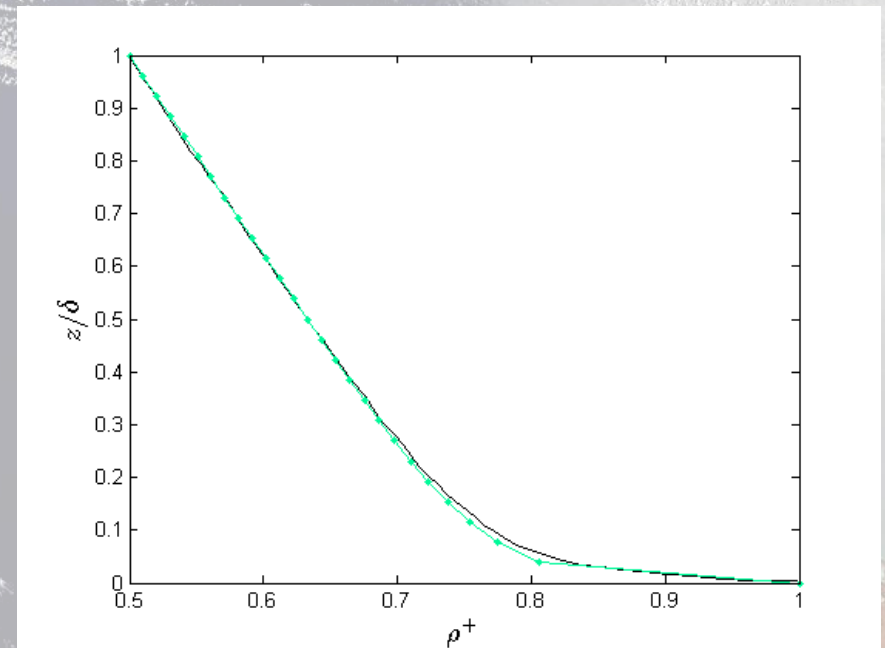
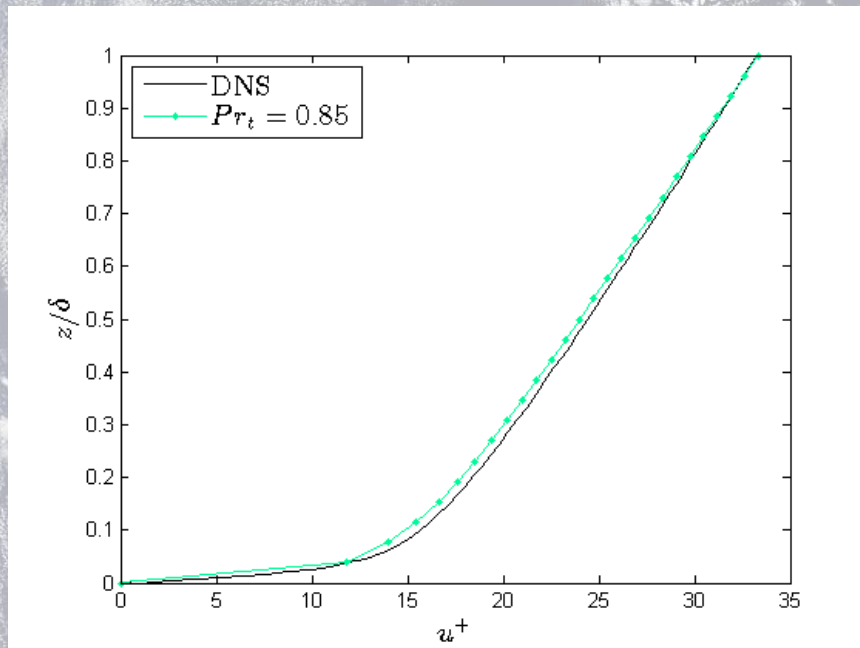
- $Ri_\tau = 167$





# Couette Flow Results

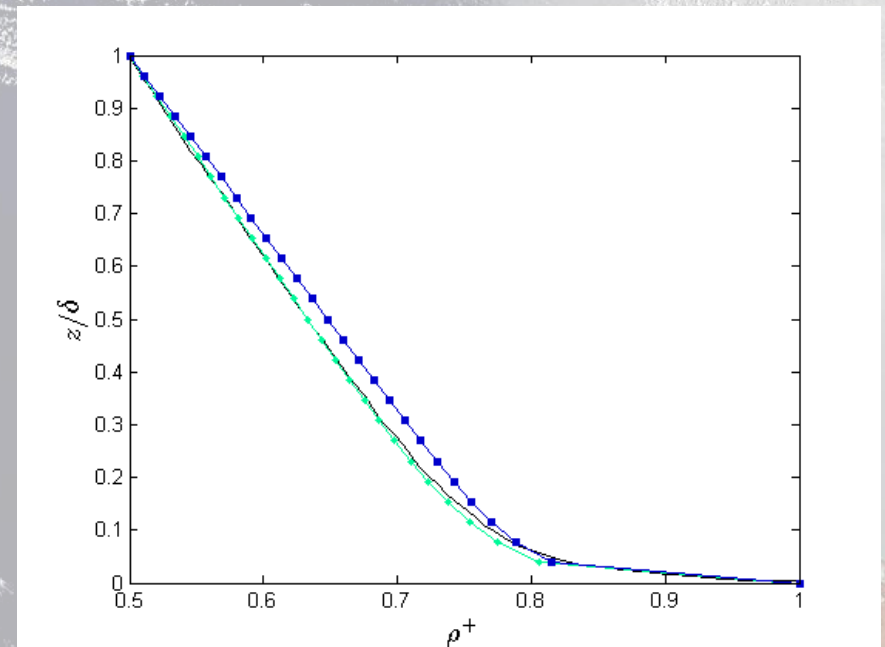
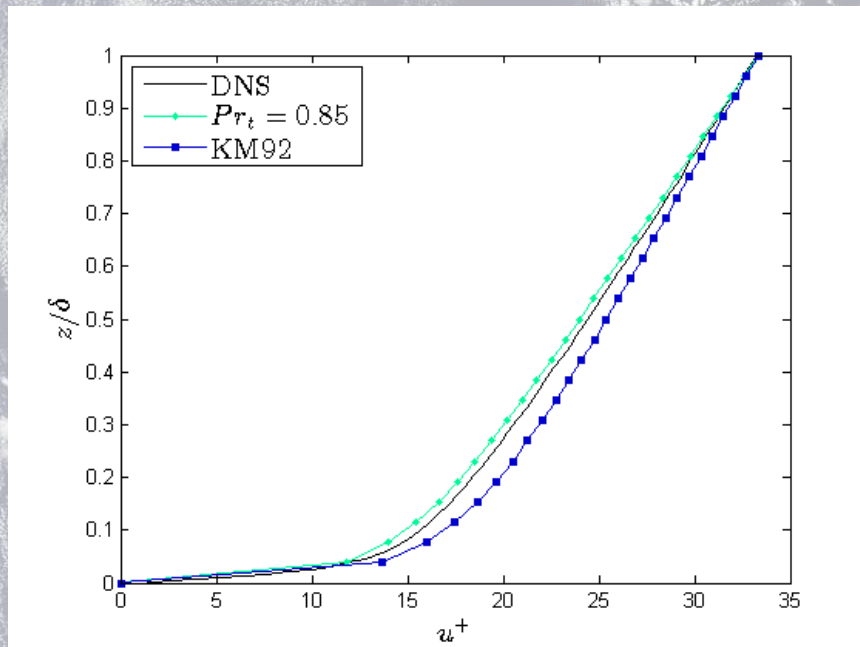
- $Ri_\tau = 167$





# Couette Flow Results

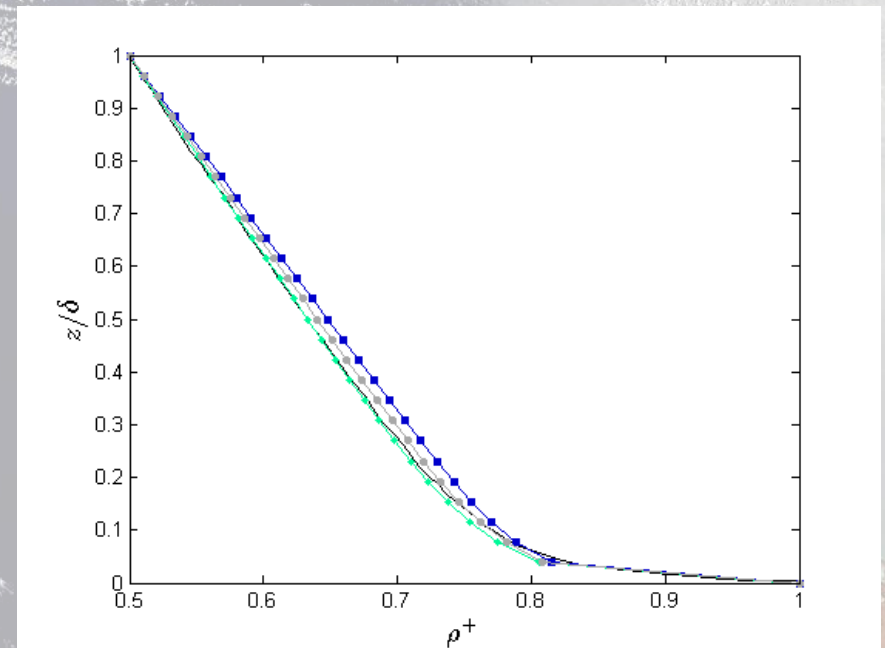
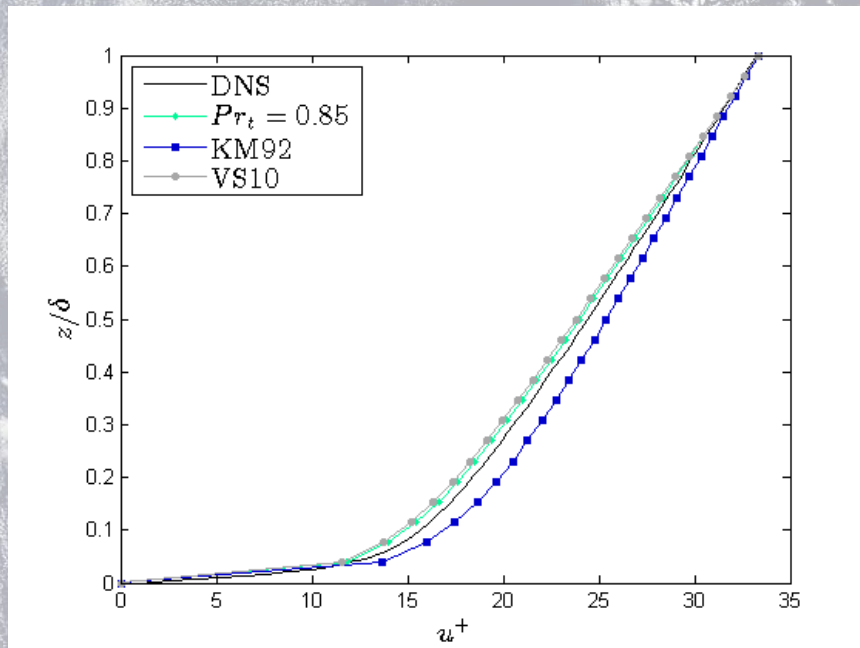
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# Couette Flow Results

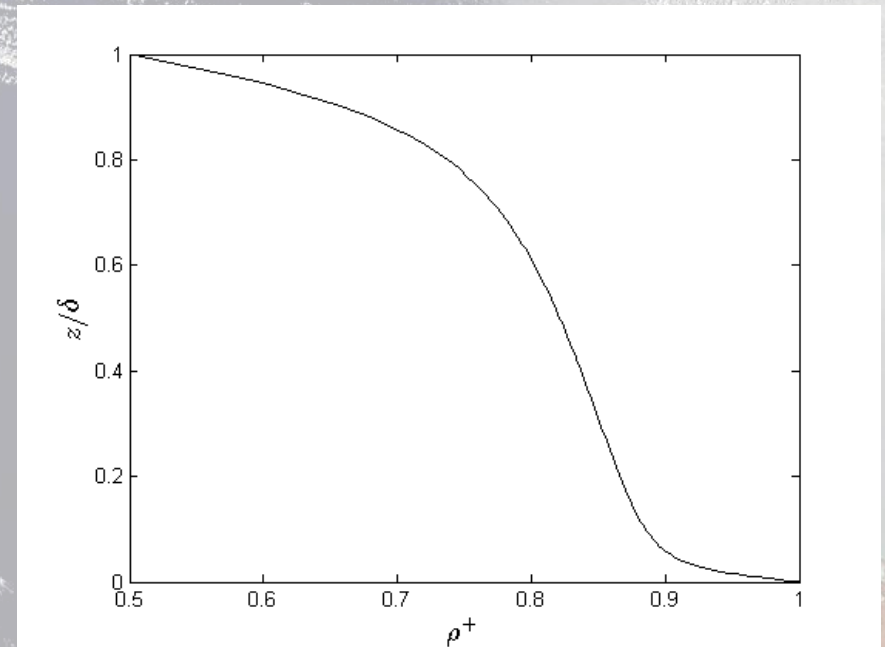
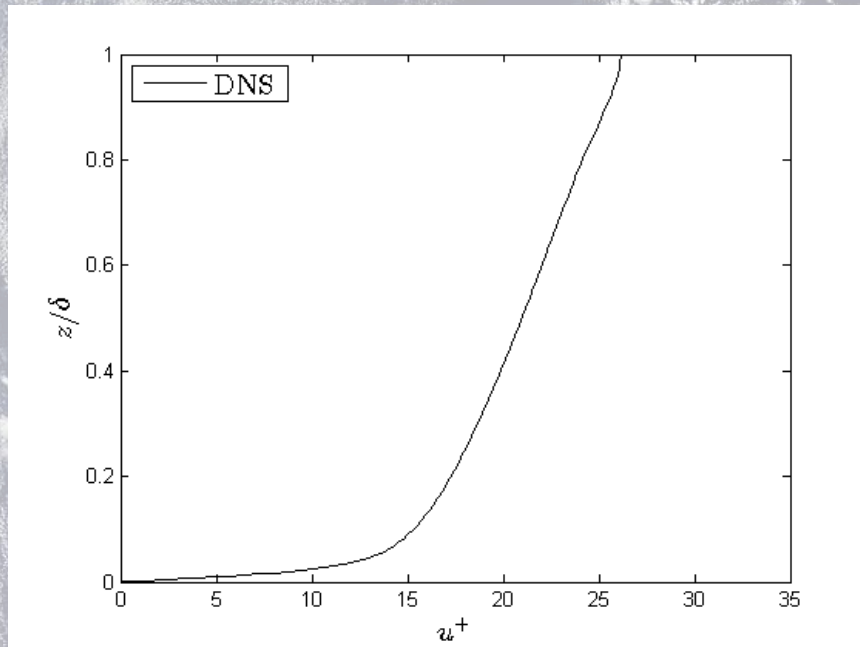
- $Ri_\tau = 167$





# Channel Flow Results

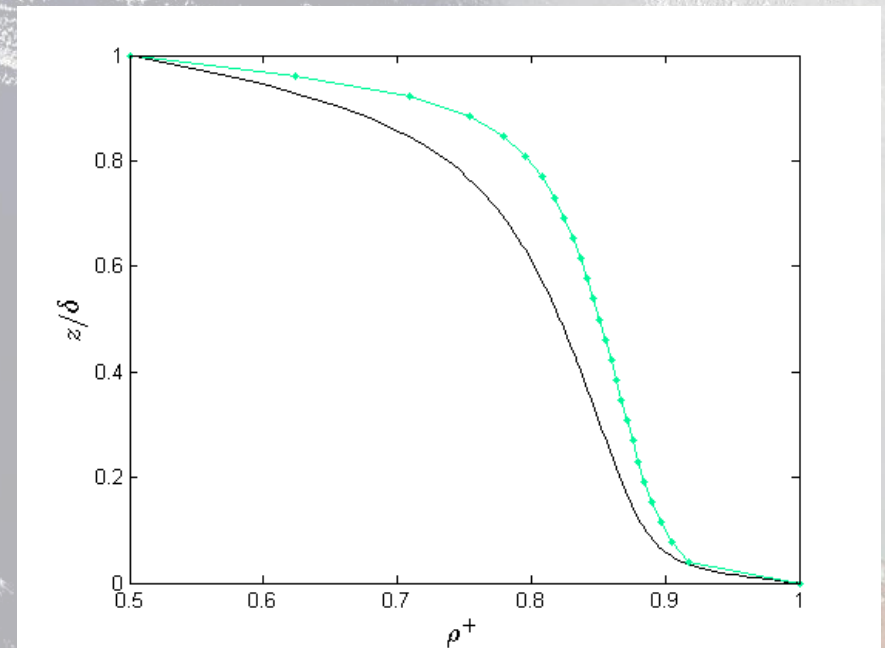
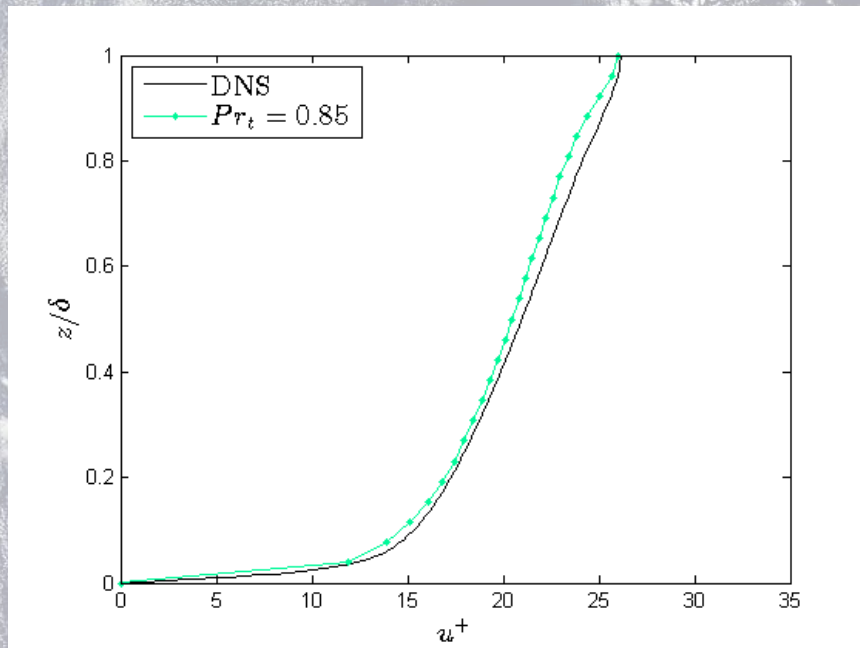
- $Ri_\tau = 60$





# Channel Flow Results

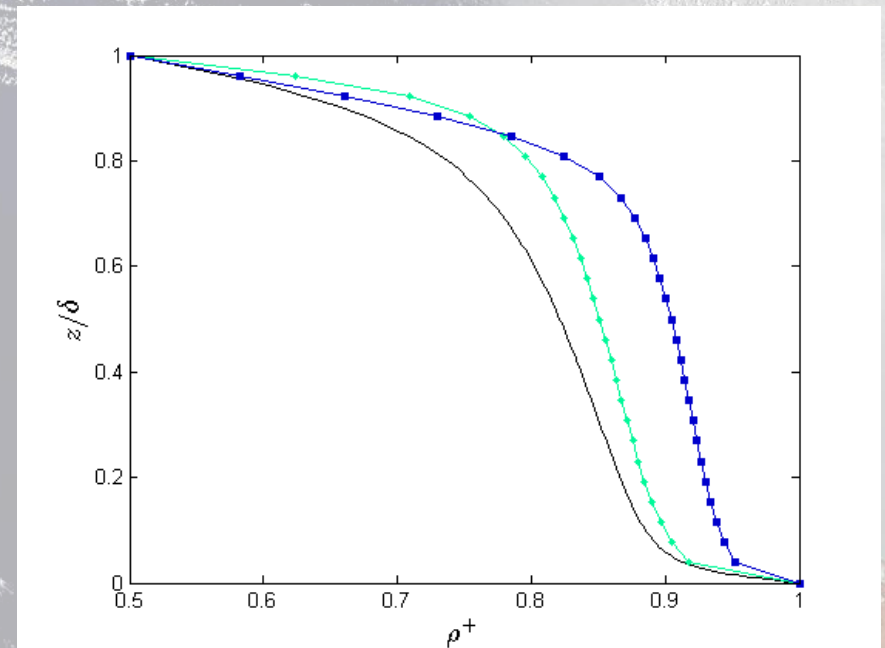
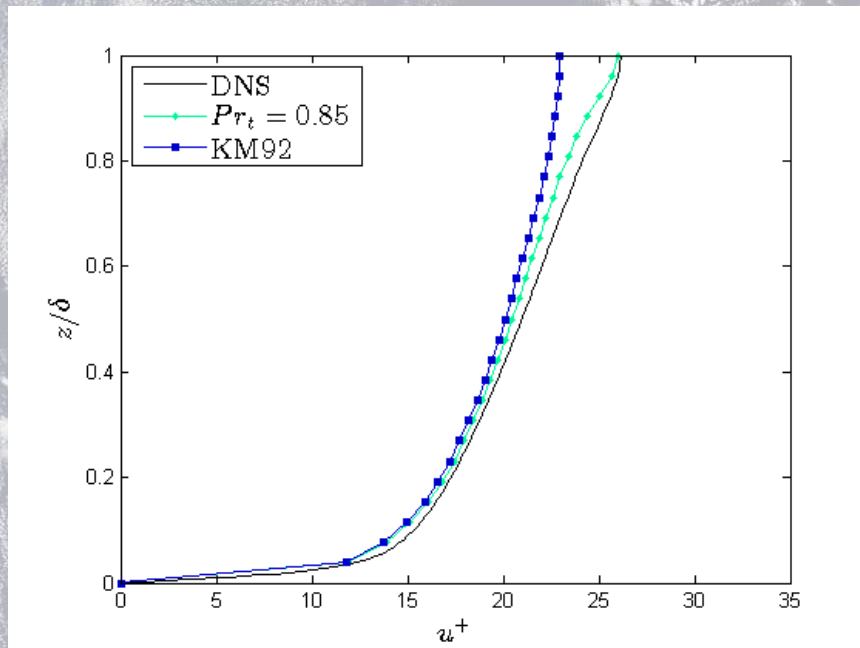
- $Ri_\tau = 60$





# Channel Flow Results

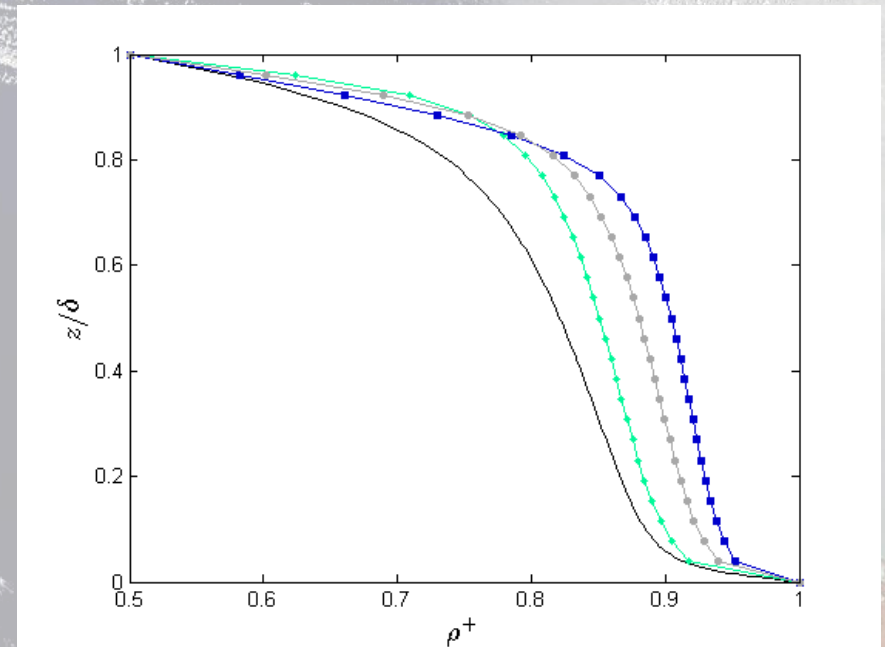
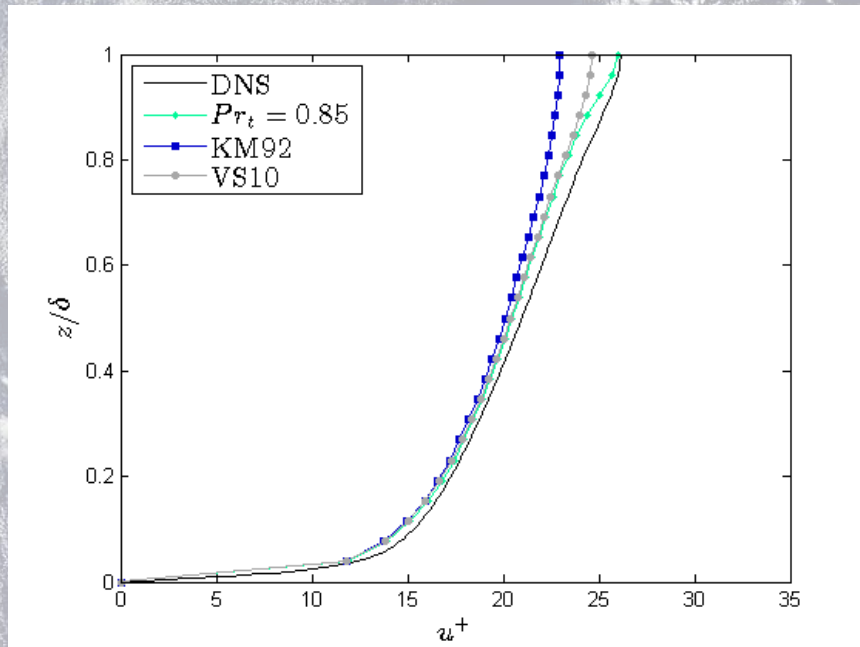
- $Ri_\tau = 60$





# Channel Flow Results

- $Ri_\tau = 60$



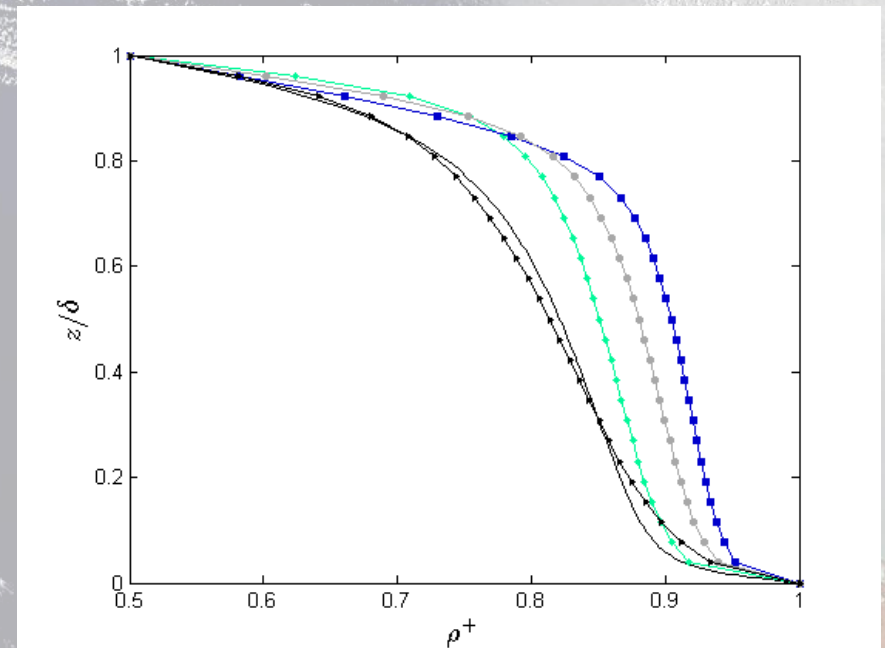
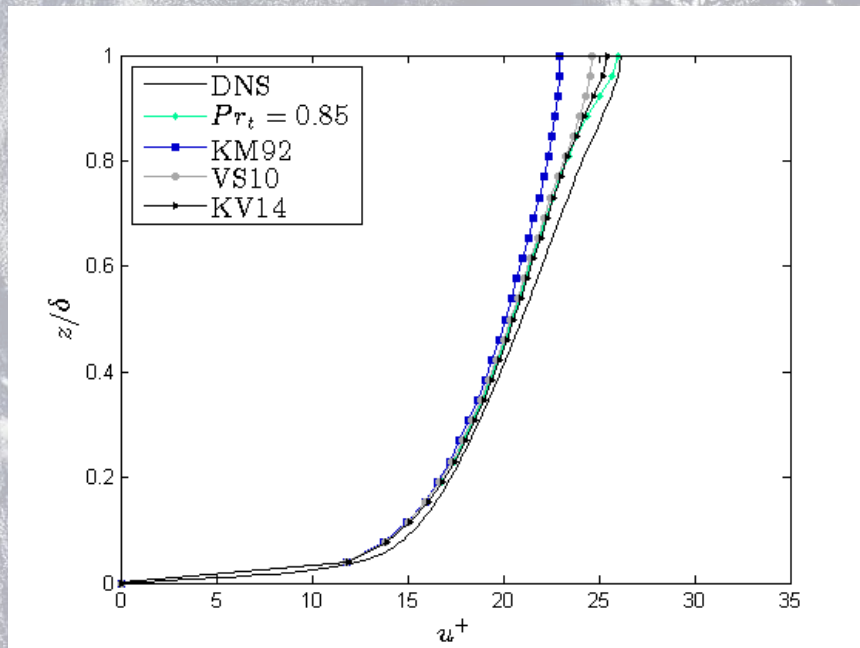


# Channel Flow Results

- All  $Pr_t$  formulations overestimate mixing of the mean density field
  - $Pr_t$  formulation of Karimpour & Venayagamoorthy (2014) for wall-bounded flows
  - Linear stress distribution in a channel flow
    - $\tau = \tau_w \left(1 - \frac{z}{D}\right)$
  - Unstratified  $Pr_t$  in a wall bounded flow
    - $Pr_{t0} = \left(1 - \frac{z}{D}\right) Pr_{twd0} + Pr_{t0} \approx 1.1$
  - The turbulent viscosity in the log-law region is given by
    - $\nu_t = \frac{u_t^2}{S} \left(1 - \frac{z}{D}\right)$
  - Further assumptions for the log-law region in a wall-bounded flow and subsequent substitutions results in
    - $Pr_t = \left(1 - \frac{z}{D}\right) \frac{Ri}{Ri_f} + \left(1 - \frac{z}{D}\right) Pr_{twd0} + Pr_{t0} \quad (KV14)$
    - Where  $Ri_f = 0.25[1 - \exp(-\gamma Ri)]$

# Channel Flow Results

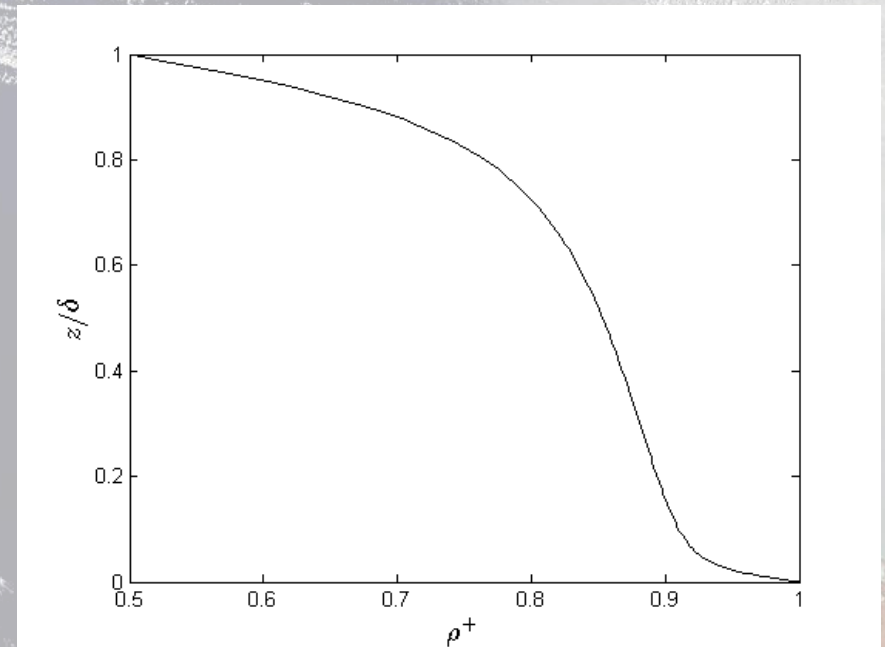
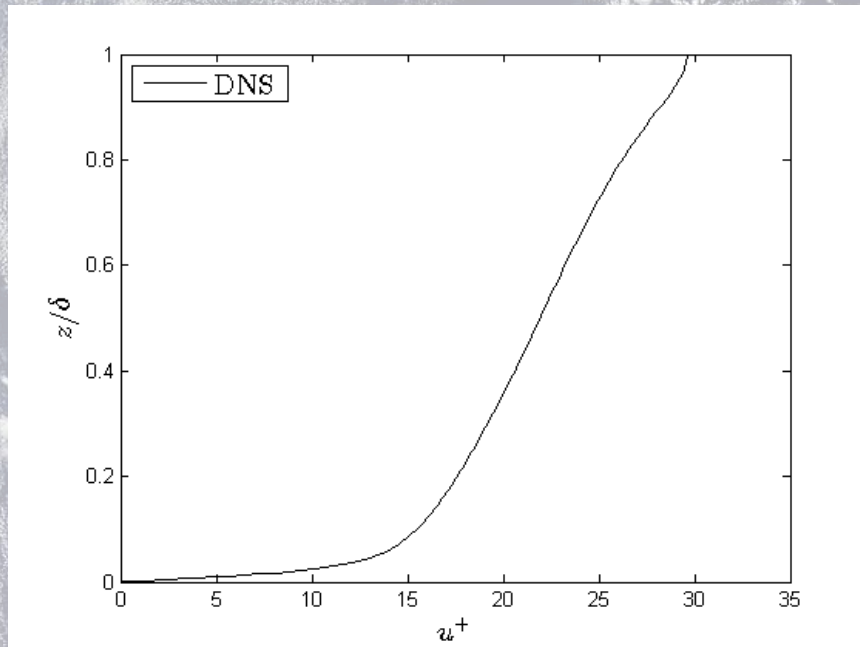
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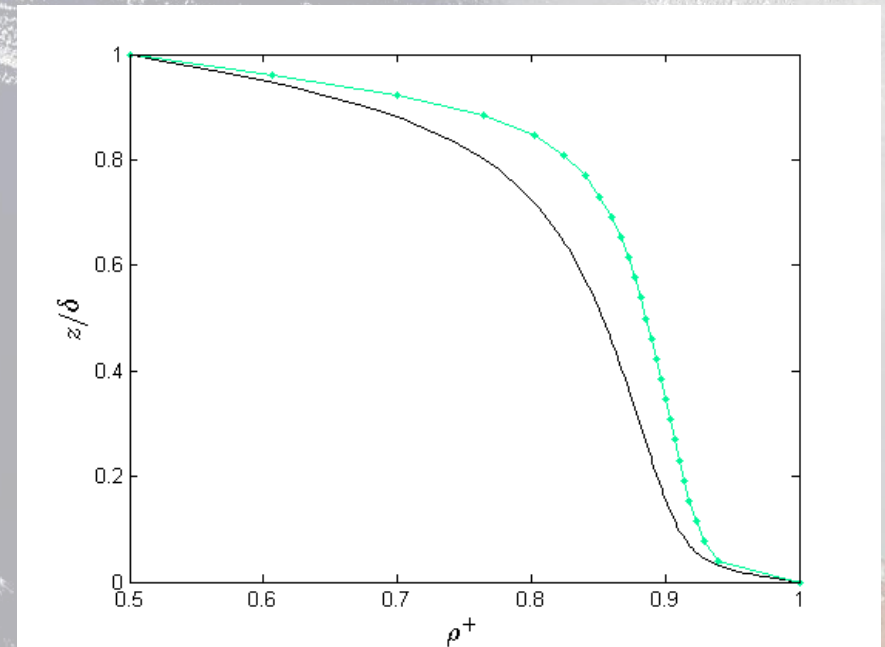
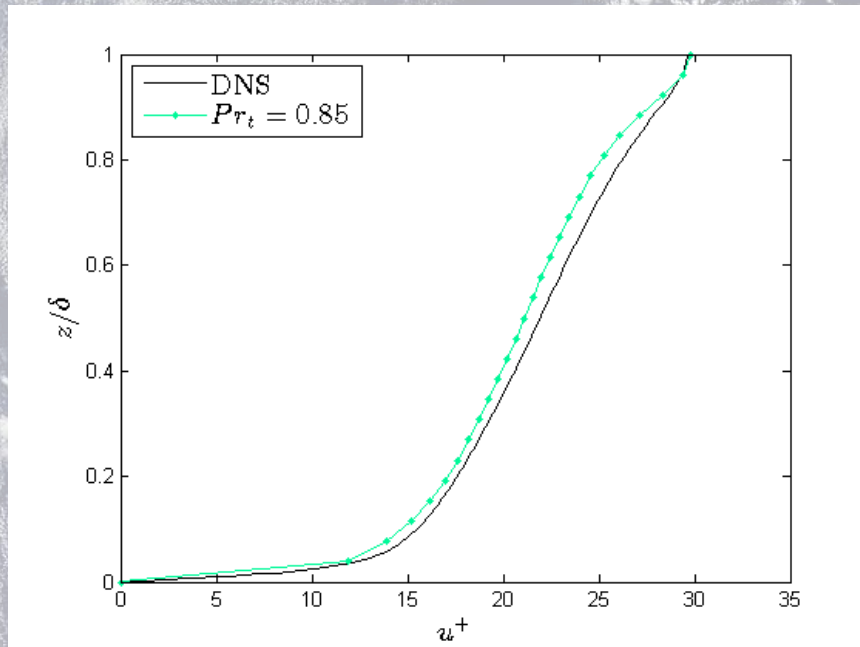
# Channel Flow Results

- $Ri_\tau = 120$



# Channel Flow Results

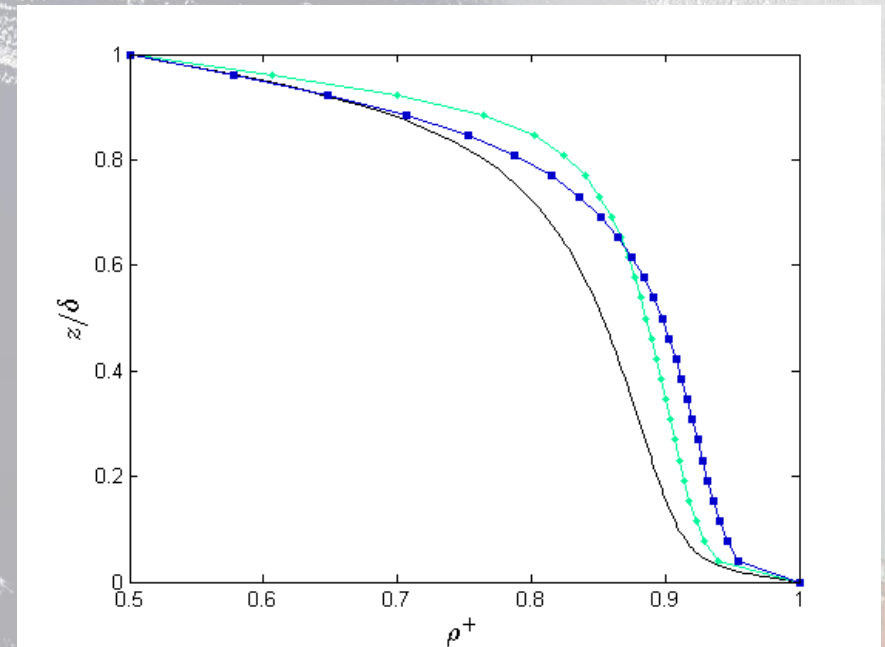
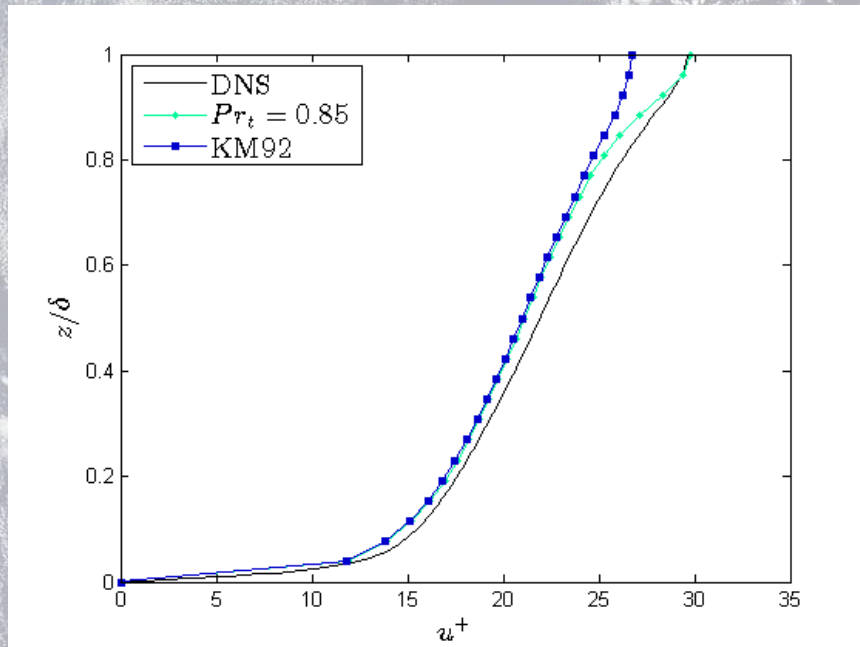
- $Ri_\tau = 120$





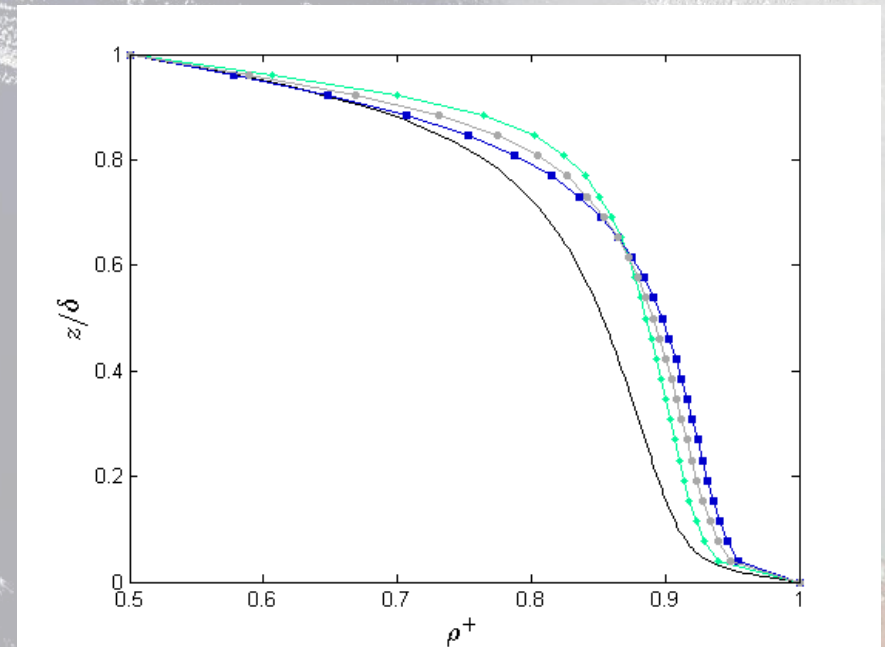
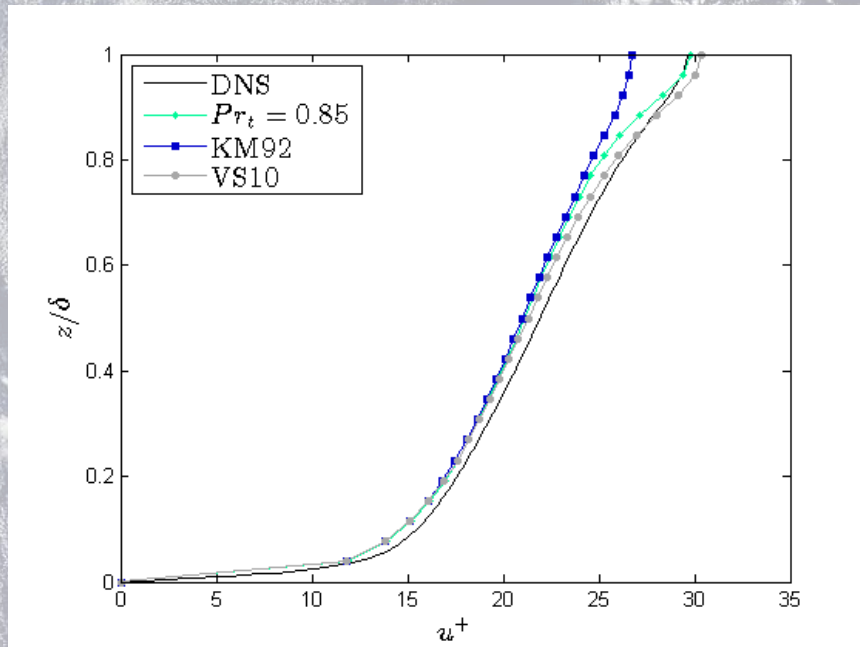
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- $Ri_\tau = 120$



# Channel Flow Results

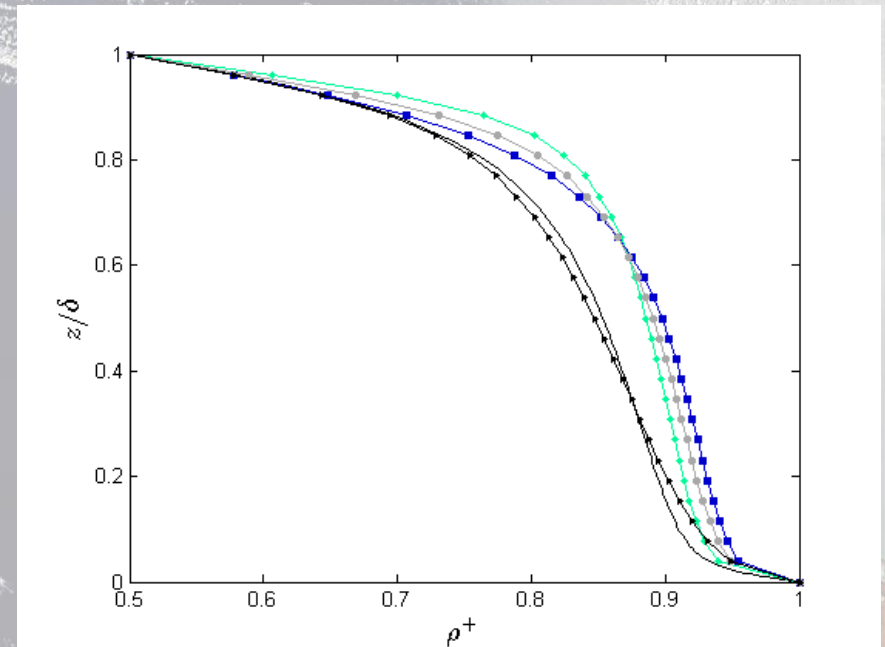
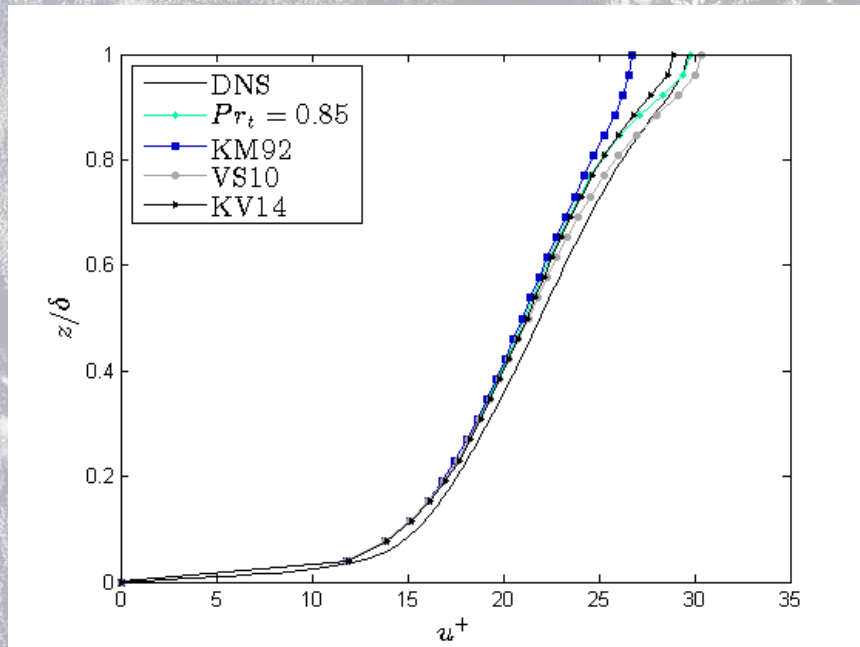
- $Ri_\tau = 120$





# Channel Flow Results

- $Ri_\tau = 120$



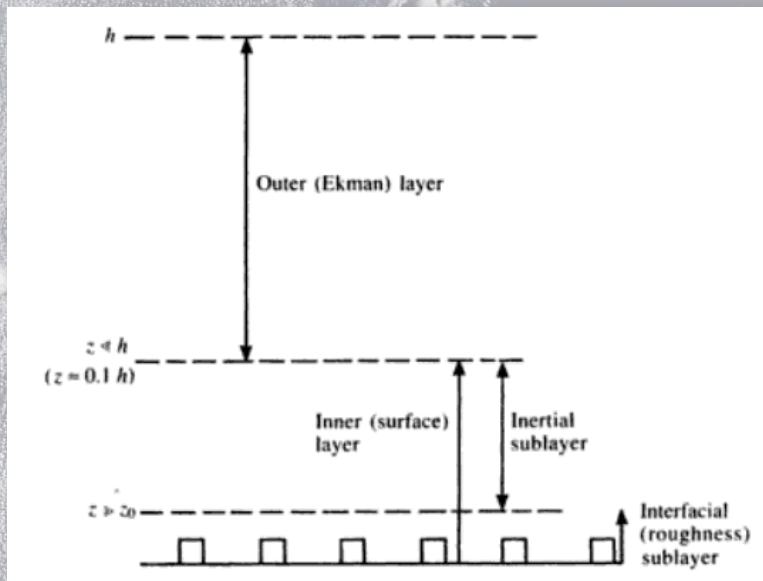
# Conclusions

- In stably stratified flows, mean velocity and density fields are evaluated
  - Accurate prediction of mean density proves more difficult than mean velocity
- For Couette flow, constant and homogeneous  $Pr_t$  formulations predict mean velocity and density fields well
- For channel flow, constant and homogeneous  $Pr_t$  formulations predict mean velocity well but over-predict mixing in the mean density field.
  - An inhomogeneous, wall-bounded  $Pr_t$  formulation significantly improves the prediction of mean density

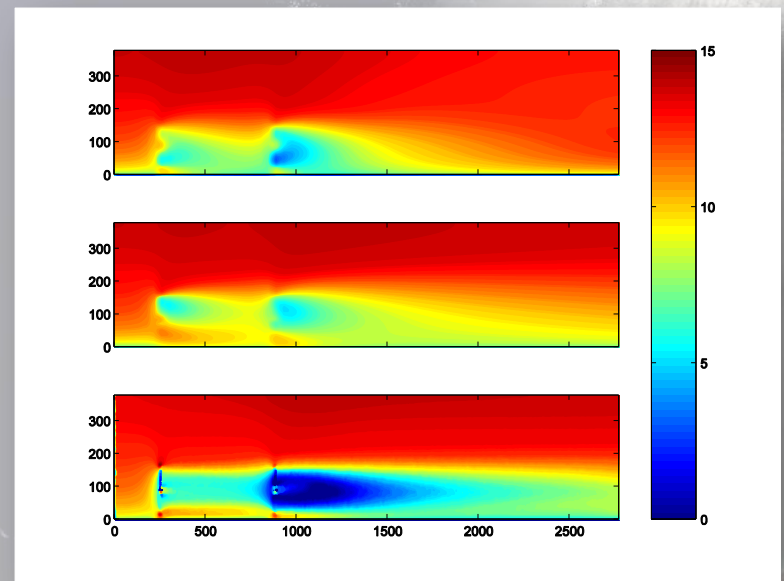


# Future Research

- Moving towards the stable ABL and wind turbine interactions
- Coriolis force (Ekman layer), surface roughness



Garratt (1992)



NREL 5MW Turbine Interactions  
under neutral conditions



# Questions?

- **Acknowledgements:**
  - M. Garcia-Villalba and J.C. del Alamo for providing their detailed post-processed DNS data of channel flow simulations
- Photo credit: Jacques Descloitres, MODIS Rapid Response Team, NASA/GSFC
  - **ATMOSPHERIC GRAVITY WAVES AND INTERNAL WAVES OFF AUSTRALIA**